

ment In the case of a uniform beam where damping is a function of displacement, Eq (4) becomes

$$\tan(\alpha_2 - \alpha_1) = \frac{\omega k(c_1 - c_2)}{2k^2 + \omega^2 c_2(c_1 + c_2)} \quad (5)$$

Hence, a massless beam may have a phase shift

Effect of Aerodynamic Damping on Flutter of Thin Panels

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THIS note discusses the role of aerodynamic damping in determining the flutter boundary for thin panels, primarily within the framework of piston theory aerodynamics

In the early work on the flutter of unstressed, flat, rectangular panels, it was shown that the neglect of aerodynamic damping was a very good approximation for panels of moderate to large aspect ratios (see, e g, Refs 1 and 7) Thus, the flutter boundary could be described in terms of the now well-known parameter,

$$\lambda = 2qL^3/(M^2 - 1)^{1/2}D$$

The omission of aerodynamic damping was equivalent to setting the mass ratio

$$\mu = \rho L / \rho m h$$

equal to zero

Subsequent to this work, the approximation $\mu = 0$ has been used in determining the stability boundaries for numerous configurations Here we would like to note that there are situations where this approximation may lead to inaccurate results This is usually associated with flutter involving modes of nearly identical frequency but weak aerodynamic coupling

1 Cylindrical Shell

As discussed in Ref 2, for the cylindrical shell there are two types of flutter modes of interest One of these is a coupling of the first two axial modes at a large circumferential mode number For this type of flutter mode the approximation is still a good one (under the assumption of piston theory aerodynamics) The other flutter mode of interest is an axisymmetric coupling of two higher axial modes For this type of flutter mode aerodynamic damping has a very pronounced effect on the flutter boundary In addition, flutter involving other modes of nearly identical frequency, but weakly coupled aerodynamically, can be shown to be unimportant due to aerodynamic damping

2 Rectangular Panels under Shearing Loads

Recently, Easley and Luessen³ considered the problem of a rectangular, flat panel under shearing loads These shearing loads lead to the merging or crossing of the natural frequencies associated with an otherwise unstressed panel, and flutter boundaries would be obtained with sharp minimums in the flutter dynamic pressure for certain values of the shearing loads for $\mu = 0$ The inclusion of aerodynamic damping removes these unrealistic "dips"

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3 Rectangular Panel Aligned at an Arbitrary Angle to the Flow

As a final example, the case of a flat, rectangular panel aligned at an arbitrary angle to the airflow is considered This problem has been studied previously in Refs 3-5 Recent calculations by the present authors including aerodynamic damping have led to the results shown in Fig 1 This figure gives the flutter parameter λ as a function of flow angle ϕ for two values of μ/M ; $\mu/M = 0.1$ was chosen as being representative of practical values of the mass ratio As may be seen, the addition of aerodynamic damping changes the character of the curve markedly In particular, the rapid variation of λ in the neighborhood of $\phi = 0$ is no longer present for $\mu/M = 0.1$

Fortunately, the inclusion of the aerodynamic damping term in the piston theory (or quasi-steady theory) is relatively simple for panels of uniform mass distribution For these panels a nondimensional generalized eigenvalue may be identified as

$$\theta = K^2 - i[\lambda\mu/M]^{1/2}K \quad (1)$$

where

$$K^2 = \rho m h L^4 \omega^2 / D$$

is a nondimensional, complex panel frequency Equating real and imaginary parts of Eq 1 and specifying the flutter condition, $K_{im} = 0$, gives

$$K_R = \pm(\theta_R)^{1/2} \quad (2)$$

$$\mu/M = (1/\lambda)(\theta_1^2/\theta_R) \quad (3)$$

Strictly speaking, for the quasi-steady theory, μ/M should be replaced by

$$[\mu/(M^2 - 1)^{1/2}][(M^2 - 2)/(M^2 - 1)]^2$$

It is to be noted that θ is only a function of λ (for a given panel geometry) Thus the mathematical problem is reduced to finding $\theta = \theta(\lambda)$ After θ has been determined, Eqs (2) and (3) may be used to determine flutter frequency and mass ratio, respectively This is essentially the stability parabola concept of Movchan⁶ and Houbolt⁷ Viscous structural damping also may be included by an analogous procedure

Finally, it should be mentioned that there are other aerodynamic damping effects which piston theory is incapable of predicting Foremost among these is the possibility of single degree of freedom flutter when the aerodynamic damping becomes negative at low supersonic Mach numbers For this type of instability, λ is no longer an appropriate parameter

Lock and Fung⁸ have investigated this phenomenon experimentally and theoretically for a nominal two-dimensional panel More recent work on the subject has shown that it exists over a limited range in Mach number which decreases

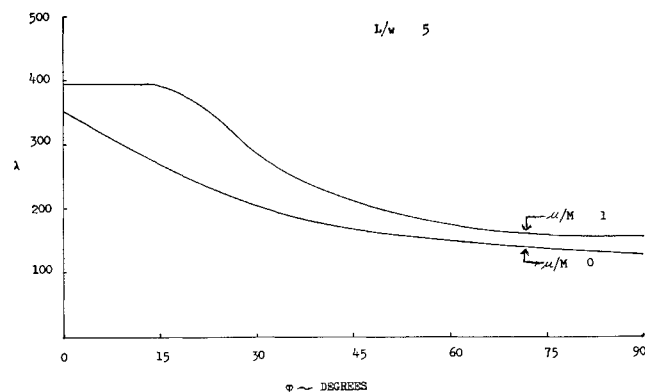


Fig 1 λ vs ϕ

with increasing panel length to width ratio. The occurrence of single degree of freedom flutter, its experimental definition and theoretical prediction, remains one of the important problems in panel flutter. A detailed discussion is not possible here, but the reader is referred to Refs 8-11 for the literature on the subject.

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Optical-Acoustic Effects in Solid Films

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Acoustics effects of light radiation on solid film are noted, and a measurement method is described. Acetylene soot, camphor soot, flat black paper, glass, white paper, aluminum, and blackbody cavities were investigated. Proposed uses are noted.

OPTICAL-ACOUSTIC effects were noted in gases in 1881. Recently, several Soviet investigators were concerned with different spectral regions and measurement methods in gases.²⁻⁶ In some of our studies on diffusivity, similar optical-acoustic effects have been found in solid films. In the system used in this investigation (Fig. 1) the specimens were irradiated with a xenon gas phototube which puts out 1 megalumen for 1 msec. The acoustic pickup was a microphone with a flat response from 60 to 13,000 cps.

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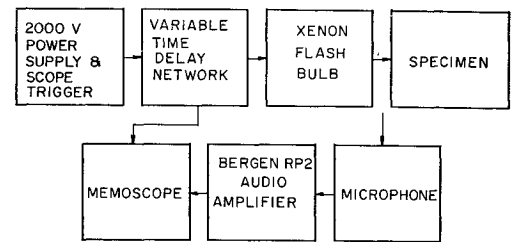


Fig. 1 Flash and audio system

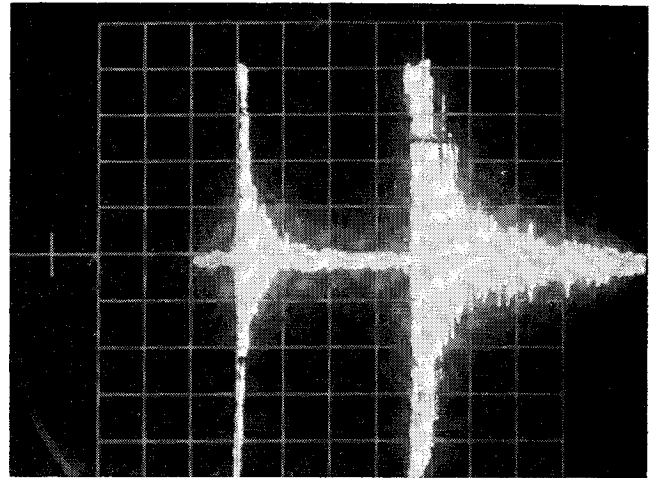


Fig. 2 Photograph of scope trace

Very sharp audible pulses were noted from acetylene soot covering on both electrical conductors and nonconductors. Less sharp reports were obtained from camphor soot and a very small noise from the flat black paper. White paper and various transparent plastics gave no report at all. The report given off by the black materials apparently is due to air motion following energy absorption.

In the use of uncoated aluminum sheet, no effect was noted for plates over 1 cm in thickness. Very thin aluminum sheets gave out a high pitched tone when irradiated. This effect appears to be due to light radiation pressure. Figure 2 shows the scope trace for acetylene soot. The left-hand pulse is relay noise obtained by closing the relay when the specimen is not irradiated, and the right-hand grouping is due to the combination of this same relay and the audible report due to energy absorption. This is a second sweep obtained when the horizontal sweep is moved and the specimen is irradiated. The sweep speed was 20 msec per division, and the vertical scale was 1 v per division. Reversing the acetylene coated glass so that the uncoated side is to the light stops the noise, thus indicating that the effect is beyond 2.7μ because the glass is not transparent beyond that point.

In this case, the light pressure or the formation of gas between the glass and the coating was sufficient to blow the coating off of the glass. Measurements from force transducers behind the specimen were inconclusive since several types of transducers appeared to be sensitive to the electrical transients in the circuit. Measurements of blackbody cavities gave results similar to the acetylene soot.

It is proposed that the order of magnitude of thermal absorptivity of nearly black specimens may be evaluated by a comparison of the magnitude of the acoustic energy released upon irradiation with the flash bulb. Extensive study would appear to be required if high accuracy were to be obtained.

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